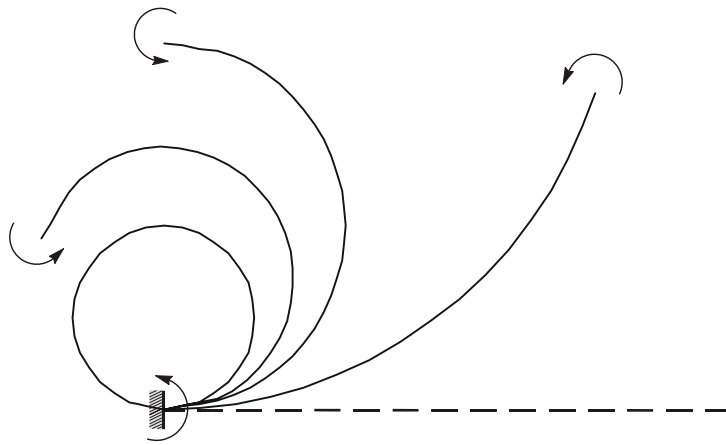


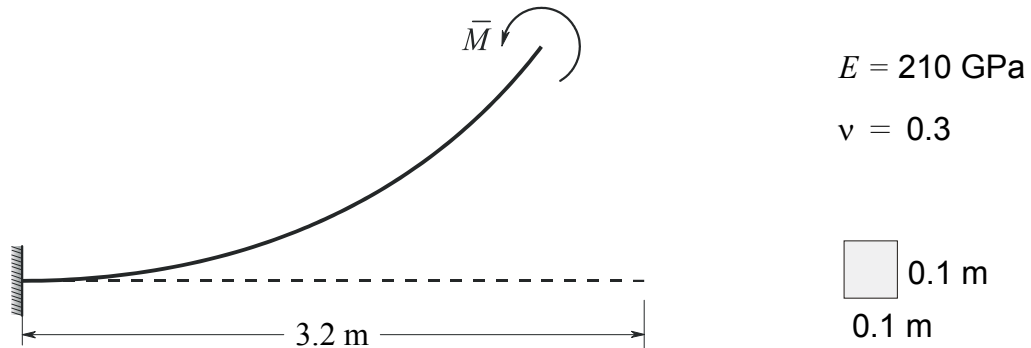
Workshop 10

Nonlinear Static Analysis of a Cantilever Beam



This example uses a simple cantilever beam model to demonstrate the geometric nonlinear effects of large rotations. A linear solution method would not be adequate for this case as equilibrium can only be satisfied in the deformed configuration due to the high applied loading. The example is extensively studied in the literature to demonstrate the efficiency of numerical methods and the large rotation capability of beam, plate and shell elements.

Model Description



The model consists of a cantilever beam: 3.2 m long, divided into 15 elements, subjected to an end moment of $\bar{M} = 3436117 \text{ N m}$ at the free end, which forces the beam to curl into a complete circle. The beam has a 0.1 m square cross section.

Exercise Procedure

1. Start up **MSC/NASTRAN for Windows 4.5** and begin to create a new model.

Double click on the icon for the **MSC/NASTRAN for Windows V4.5**.

On the *Open Model File* form, select **New Model**.

Turn off the workplane:

Tools / Workplane (or F2) / ☐ Draw Workplane / Done

View / Regenerate (or Ctrl G).

2. Create a material called **mat_1**.

From the pulldown menu, select **Model / Material**.

Title

Young's Modulus

Poisson's Ratio

Select **OK / Cancel**.

NOTE: In the *Messages Window* at the bottom of the screen, you should see a verification that the material was created. You can check here throughout the exercise to both verify the completion of operations and to find an explanation for errors which might occur.

3. Create a property called **prop_1** to apply to the members of the beam.

From the pulldown menu, select **Model / Property**.

Title

Material

Elem / Property Type

Change the property type from **Plate** element (default) to **Beam** element.

Line Elements

Beam

Select **OK**.

To select the cross-sectional shape, click on **Shape**:

Shape

Rectangular Bar

H

0.1

Width

0.1

Orientation Direction (y)

✓ Up

Select **OK / OK / Cancel**.

4. Create the beam model (15 elements along the beam).

We will do a bit different from Workshops 7 and 8.

From the pulldown menu, select **Geometry / Curve-Line / Coordinates**.

	X:	Y:	Z:					
<i>Point 1</i>	<table border="1"><tr><td>0</td></tr></table>	0	<table border="1"><tr><td>0</td></tr></table>	0	<table border="1"><tr><td>0</td></tr></table>	0	<table border="1"><tr><td>OK</td></tr></table>	OK
0								
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OK								
<i>Point 2</i>	<table border="1"><tr><td>3.2</td></tr></table>	3.2	<table border="1"><tr><td>0</td></tr></table>	0	<table border="1"><tr><td>0</td></tr></table>	0	<table border="1"><tr><td>OK / Cancel</td></tr></table>	OK / Cancel
3.2								
0								
0								
OK / Cancel								

To fit the display onto the screen, select **View / Autoscale / Visible** (or **Ctrl A**).

Now define de mesh.

From the pulldown menu, select **Mesh / Mesh Control / Custom Size Along Curve**.

Select the curve on the screen using the mouse and, then, **OK**.

Num Elements

15

OK / Cancel

Mesh the curve.

From the pulldown menu, select **Mesh / Geometry / Curve**.

Select the curve on the screen using the mouse and, then, **OK**.

Property

prop_1

OK

Now, specify the orientation vector for the beam elements.

X:

Y:

Z:

Base

0

0

0

OK

Tip

0

1

0

Preview / OK

NOTE: In MSC/NASTRAN, the way to construct the element coordinate system is by defining an *orientation vector*, as explained in Workshop 2. The element lies on the local x axis and the moments of inertia I_y and I_z are related to the bending about the local y and z axes, respectively. So, be certain that you understand the assumed beam orientation.

5. Create the model constraints.

Before creating the appropriate constraints, a constraint set needs to be created.

Do so by performing the following:

Model / Constraint / Set

Title

constraint_1

Select **OK**.

Now, define the relevant constraint for the model.

Model / Constraint / Nodal

Select **Node 1 / OK**.

On the *DOF* box, select all 6 boxes (or just *fixed*)

<input checked="" type="checkbox"/>	TX	<input checked="" type="checkbox"/>	TY	<input checked="" type="checkbox"/>	TZ
<input checked="" type="checkbox"/>	RX	<input checked="" type="checkbox"/>	RY	<input checked="" type="checkbox"/>	RZ

Select **OK** / **Cancel**.

Notice that the constraint appears on the screen at Node 1, fixing the 1, 2, ..., 6 directions (corresponding to TX, TY, TZ, RX, RY and RZ).

6. Create the model loading.

Like the constraints, a load set must first be created before creating the appropriate model loading.

Model / Load / Set (or **Ctrl F2**)

Title

Select **OK**.

Now, define the 3436117 N m applied moment.

Model / Load / Nodal

Select **Node 16** / **OK**.

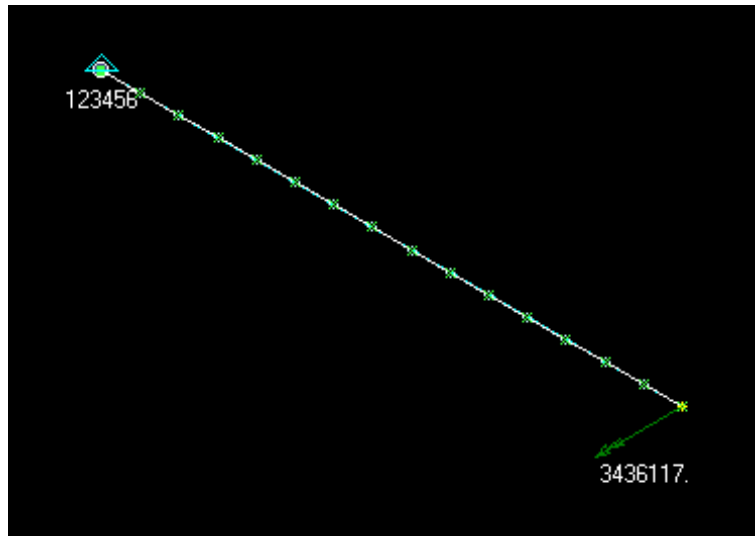
Highlight **Moment**

Load **MZ** ☒

Select **OK** / **Cancel**.

To obtain a better view, do the following:

View / Rotate (or **F8**) / **Isometric** / **OK**.



7. Set the nonlinear solution parameters.

Model / Load / Nonlinear Analysis

Solution Type

☒ **Static**

Click the **Defaults** button to apply the default values for the nonlinear solution control parameters.

The number **10** is a reasonable value for the **Number of Increments** at this analysis.

If results are desired at every increment, then the **Output Control Intermediate** box should be changed to **ALL**.

Load Set Options for Nonlinear Analysis

Load Set 1 load_1

Solution Type

☐ Off ☒ Static ☐ Creep ☐ Transient

Basic

Number of Increments 10

Time Increment 0.

Max Iterations / Step 25

Stiffness Updates

Method 0..Default

Iterations Before Update 5

Output Control

Intermediate 3..ALL

Output Every Nth Step 1

Convergence Tolerances

☐ Displacement 0,001

☒ Load 0,001

☒ Work 1,E-7

Solution Strategy Overrides

☐ Arc-Length Method

☒ None or Advanced Overrides

☐ Full Newton-Raphson

☐ Modified Newton-Raphson

☒ Line Search

☒ Quasi-Newton

☒ Bisection

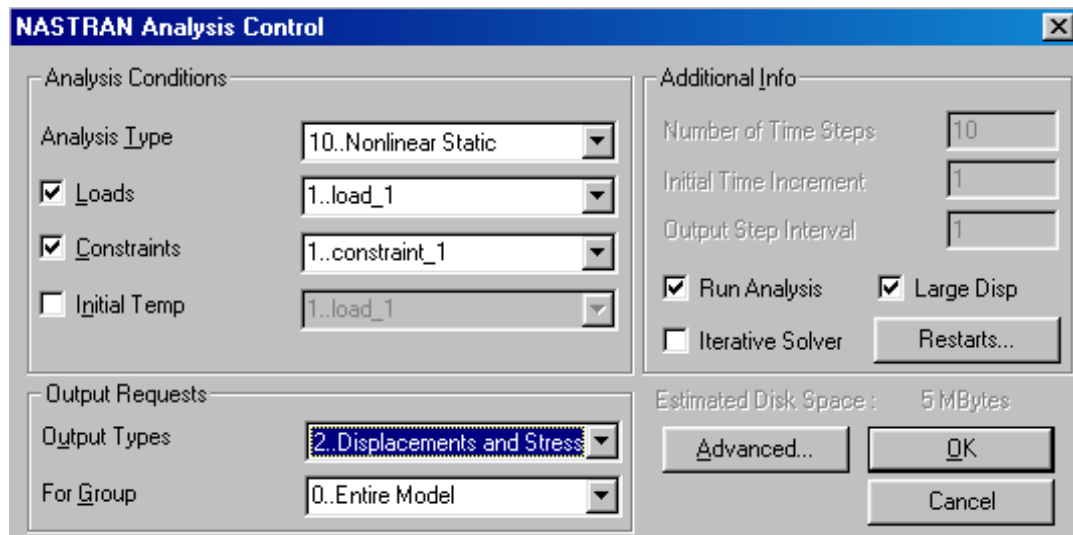
Advanced... Copy... Defaults... OK Cancel

Now click **OK** to accept the chosen selections.

It is possible to change the nonlinear solution parameters by using **Solution Strategy Overrides**, for instance, but it is not usually necessary. For certain problems, however, the default settings may not lead to a solution coverage.

8. Run the analysis.

File / Analyze



Analysis Type

Nonlinear Static

Loads

☒ **load_1**

Constraints

☒ **constraint_1**

☒ **Run Analysis**

☒ **Large Disp**

Output Types

☒ **Displacements and Stresses**

Select **OK**.

When asked if you wish to save the model, respond **Yes**.

Be sure to set the desirable working directory.

File Name

work_10

Select **Save**.

When the MSC/ NASTRAN manager is through running, MSC/ NASTRAN for Windows will be restored on your screen, and the *Message Review* form will appear.

To read the messages, you could select **Show Details**. Since the analysis ran smoothly, we will not bother with the details this time. Then select **Continue**.

When asked: *OK to Read Nonlinear Stresses and Strains?* Respond **Yes**.

9. Processing the results.

View / Select (or **F5**)

Deformed Style

Deform

Deformed and Contour Data

Output Set

Case 20 Time 1

Output Vectors / Deformation

Total Translation

Select **OK** / **OK**.

Notice that the automatic iteration control process of MSC/NASTRAN for Windows has automatically added some additional load increments on top of the 10 selected in the nonlinear parameter settings. In the analysis phase, 10 iterations clearly were not enough to satisfy the convergence criteria so the program automatically added a sufficient number to obtain a solution.

View / Options (or **F6**)

Category



PostProcessing

Highlight **Deformed Style**

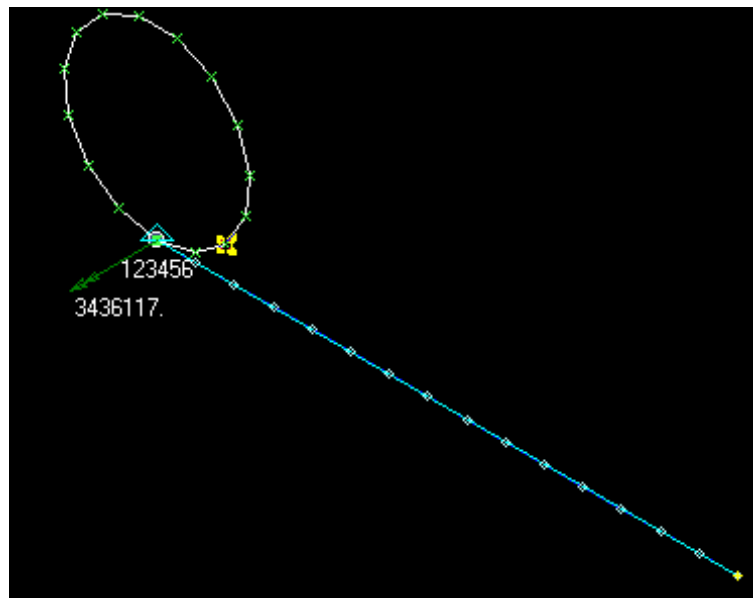
% of Model (Actual) ☐

Scale Act

1

OK

This ensures that the deformed shape is set to the actual scale.



The above shows the actual deformation of the cantilever due to the applied end moment loading.

This concludes the exercise.

File / Save

File / Exit.